

Chapter 4: Free linear damped systems with one degree of freedom

4.1 Damping force

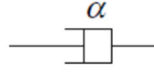
A system subject to friction is said to be damped. The simplest friction is viscous friction.

Viscous friction is of the form:

$$f = -\alpha v$$

α is a positive constant called the coefficient of friction and v is the speed of the moving body.

In mechanics, damping is schematized by:



4.2 Lagrange equation of damped systems

If there is the friction $f = -\alpha\dot{q}$, the Lagrange equation becomes:

$$\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{q}}\right) - \left(\frac{\partial L}{\partial q}\right) = -\alpha\dot{q}$$

By introducing the dissipation function $D = \frac{1}{2}\alpha\dot{q}^2$, we can write:

$f = -\alpha\dot{q} = -\frac{\partial D}{\partial \dot{q}}$. (In translation $D = \frac{1}{2}\alpha v^2 = \frac{1}{2}\alpha\dot{x}^2$. In electricity $D = \frac{1}{2}Ri^2 = \frac{1}{2}R\dot{q}^2$). The

Lagrange equation of damped systems is then written (where $q=x, y, z, q, \theta \dots$)

$$\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{q}}\right) - \left(\frac{\partial L}{\partial q}\right) = -\frac{\partial D}{\partial \dot{q}}$$

4.3 Equation of motion of damped systems

The equation of motion of linear damped systems by $f = -\alpha\dot{q}$ is of the form:

- The damped oscillation is defined as follows:

$$\ddot{q}(t) + 2\delta\dot{q}(t) + \omega_0^2 q(t) = 0$$

Where, δ is a positive coefficient and is called the damping factor. ω_0 is the proper pulsation.

$\frac{\omega_0}{2\delta} = Q$ is called the quality factor.

4.4 Solving the equation of motion

$$\ddot{q}(t) + 2\delta\dot{q}(t) + \omega_0^2 q(t) = 0$$

This equation is solved by the change of variable $q(t) = Ae^{rt} \Rightarrow \dot{q}(t) = Are^{rt} \Rightarrow \ddot{q}(t) = Ar^2e^{rt}$, the equation then becomes:

$$\ddot{q}(t) + 2\delta\dot{q}(t) + \omega_0^2 q(t) = 0$$

$$Ar^2e^{rt} + 2\delta Are^{rt} + \omega_0^2 Ae^{rt} = 0$$

$$Ae^{rt}(r^2 + 2\delta r + \omega_0^2) = 0$$

$$r^2 + 2\delta r + \omega_0^2 = 0$$

We calculate the discriminant Δ and we then obtain:

$$\Delta = (2\delta)^2 - 4\omega_0^2$$

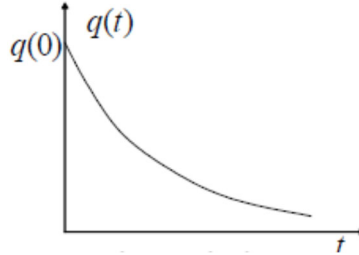
$$\Delta = 4\delta^2 - 4\omega_0^2$$

$$\Delta = 4(\delta^2 - \omega_0^2)$$

$$\sqrt{\Delta} = \sqrt{4(\delta^2 - \omega_0^2)} = 2\sqrt{(\delta^2 - \omega_0^2)}$$

There are three types of solution:

- **Case where the system is strongly damped:** $\Delta > 0 \Rightarrow \delta > \omega_0$ and $Q < 0.5$



The solution of the differential equation is written as follows:

$$q(t) = A_1 e^{r_1 t} + A_2 e^{r_2 t} \text{ avec } r_{1,2} = \frac{-2\delta \pm 2\sqrt{\delta^2 - \omega_0^2}}{2} = -\delta \pm \sqrt{\delta^2 - \omega_0^2}$$

$$q(t) = A_1 e^{(-\delta - \sqrt{\delta^2 - \omega_0^2})t} + A_2 e^{(-\delta + \sqrt{\delta^2 - \omega_0^2})t} = e^{-\delta t} \left(A_1 e^{(-\sqrt{\delta^2 - \omega_0^2})t} + A_2 e^{(\sqrt{\delta^2 - \omega_0^2})t} \right)$$

Where A_1 and A_2 are coefficients to be determined by the initial conditions

$$\begin{cases} q(t=0) \\ \dot{q}(t=0) \end{cases}$$

The system is said to have aperiodic motion.

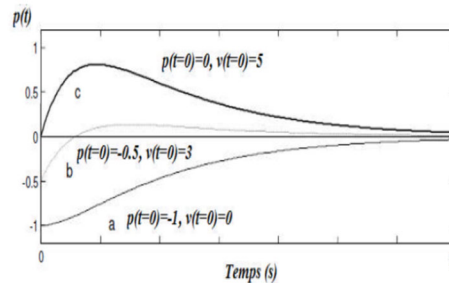
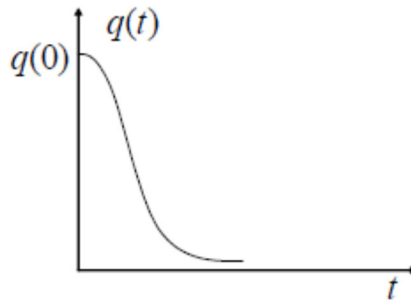


Figure 3.1 : Aperiodic damped movement

- **Case where damping is critical:** $\Delta = 0 \Rightarrow \delta = \omega_0$ et $Q = 0.5$



The solution of the equation is of the form:

$$q(t) = (A_1 t + A_2) e^{rt}$$

$$r_1 = r_2 = r = \frac{-2\delta}{2} = -\delta$$

$$q(t) = (A_1 t + A_2) e^{-\delta t}$$

Where A_1 and A_2 are coefficients to be determined by the initial conditions:

$$\begin{cases} q(t=0) \\ \dot{q}(t=0) \end{cases}$$

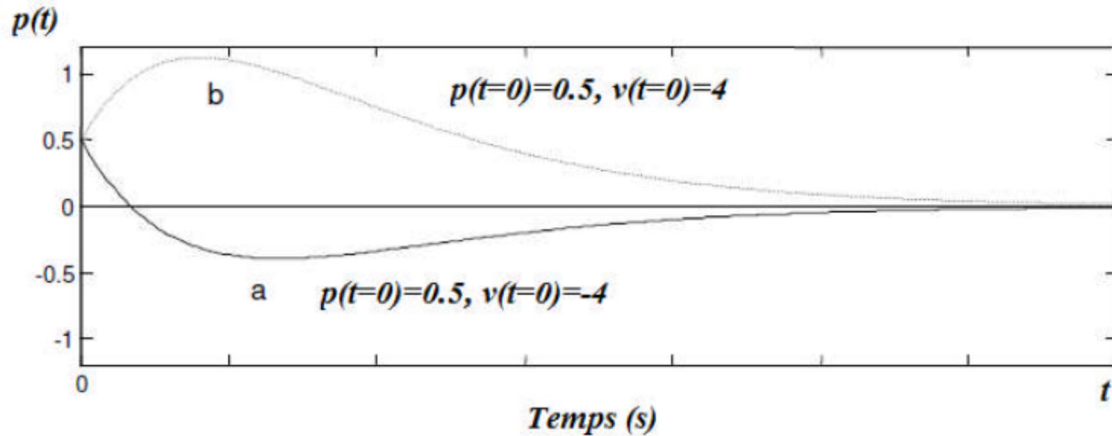
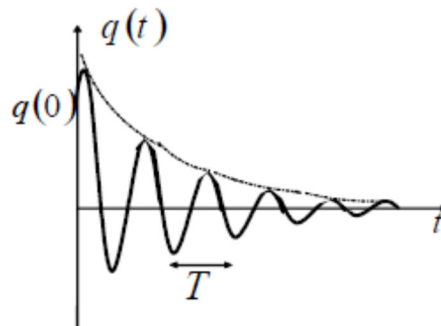


Figure 3.2 : Critical damped movement

- **Case where damping is low:** $\Delta < 0 \Rightarrow \delta < \omega_0$ et $Q > 0.5$



Two complex solutions for the characteristic equation

$$\begin{cases} r_1 = -\delta - j\sqrt{\omega_0^2 - \delta^2} \\ r_2 = -\delta + j\sqrt{\omega_0^2 - \delta^2} \end{cases}$$

The resulting motion is $q(t) = A_1 e^{-r_1 t} + A_2 e^{-r_2 t}$ that is:

$$q(t) = A e^{-\delta t} \cos(\omega t + \varphi)$$

Where, A et φ are constants to be determined by the initial conditions: $\begin{cases} q(t=0) \\ \dot{q}(t=0) \end{cases}$

The motion is said to be pseudo-periodic. $\omega = \sqrt{\omega_0^2 - \delta^2}$ is called pseudo-pulsation.

$T = \frac{2\pi}{\omega} = \frac{2\pi}{\sqrt{\omega_0^2 - \delta^2}}$ is called pseudo-period.

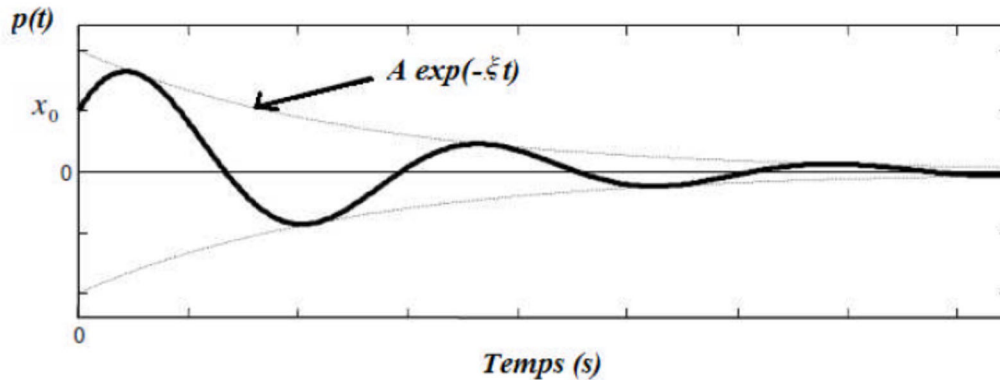


Figure 3.3: Damped oscillating motion

3.5 Logarithmic decrement

To evaluate the exponential decrease in the amplitude of the pseudo-periodic motion, we use the logarithm. The report

$D = \ln \frac{q(t)}{q(t+T)}$ or $D = \frac{1}{n} \ln \frac{q(t)}{q(t+nT)}$ is called the logarithmic decrement. Using the equation

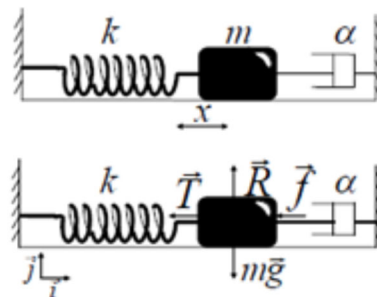
$q(t) = Ae^{-\delta t} \cos(\omega t + \varphi)$, we find $\delta = \ln \frac{Ae^{-\delta t}}{Ae^{-\delta(t+T)}} \Rightarrow D = \delta T$

It should be noted that the system undergoes a total loss of energy due to the work of the friction forces.

$$dE_T(t) = -\alpha \dot{p}(t)^2 dt = -dW_{fr} \Rightarrow \Delta E_T + \Delta W_{fr} = 0$$

Examples

a) Consider the mass-spring system. Find the equation of motion first with the Lagrangian then with PFD.



Solution

• Using the Lagrangian:

The Lagrangian is:

$$L = T - U = \frac{1}{2}m\dot{x}^2 - \frac{1}{2}kx^2.$$

$$D = \frac{1}{2}\alpha v^2 = \frac{1}{2}\alpha\dot{x}^2$$

The Lagrange equation is then written:

$$\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}}\right) - \left(\frac{\partial L}{\partial x}\right) = -\frac{\partial D}{\partial \dot{x}}$$

$$\text{with } L = T - U = \frac{1}{2}m\dot{x}^2 - \frac{1}{2}kx^2 + cte$$

$$\frac{\partial L}{\partial \dot{x}} = \frac{\partial}{\partial \dot{x}}\left(\frac{1}{2}m\dot{x}^2\right) = \frac{1}{2}m \frac{\partial}{\partial \dot{x}}(\dot{x}^2) = \frac{1}{2}m(2\dot{x}) = m\dot{x} \Rightarrow \frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}}\right) = \frac{d}{dt}(m\dot{x}) = m \frac{d}{dt}(\dot{x}) = m\ddot{x}$$

$$\frac{\partial L}{\partial x} = \frac{\partial}{\partial x}\left(-\frac{1}{2}kx^2\right) = -\frac{1}{2}k \frac{\partial}{\partial x}(x^2) = -\frac{1}{2}k(2x) = -kx$$

$$D = \frac{1}{2}\alpha v^2 = \frac{1}{2}\alpha\dot{x}^2 \Rightarrow \frac{\partial D}{\partial \dot{x}} = \frac{\partial}{\partial \dot{x}}\left(\frac{1}{2}\alpha\dot{x}^2\right) = \frac{1}{2}\alpha \frac{\partial}{\partial \dot{x}}(\dot{x}^2) = \frac{1}{2}\alpha(2\dot{x}) = \alpha\dot{x}$$

So

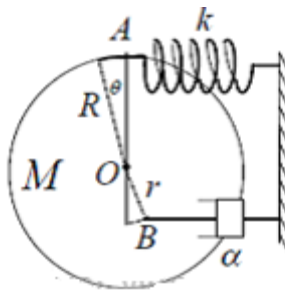
$$m\ddot{x} + kx = -\alpha\dot{x} \Rightarrow \begin{cases} \ddot{x} + \frac{k}{m}x + \frac{\alpha}{m}\dot{x} = 0 \\ \ddot{x} + \omega_0^2 x + 2\delta\dot{x} = 0 \end{cases}$$

$$\omega_0^2 = \frac{k}{m} \Rightarrow \omega_0 = \sqrt{\frac{k}{m}} \text{ et } 2\delta = \frac{\alpha}{m} \Rightarrow \delta = \frac{\alpha}{2m}$$

$$\text{Using the PFD : } \sum \vec{F} = m \vec{a} \Rightarrow m\vec{g} + \vec{T} + \vec{R} + \vec{f} = m \vec{a} \Rightarrow -kx\vec{i} - mg\vec{j} + R\vec{j} - \alpha\dot{x}\vec{i} = m\ddot{x}\vec{i}$$

$$\text{By projection on } x'Ox: -kx - \alpha\dot{x} = m\ddot{x} \Rightarrow m\ddot{x} + kx + \alpha\dot{x} = 0 \Rightarrow \ddot{x} + \frac{k}{m}x + \frac{\alpha}{m}\dot{x} = 0$$

b) Consider the disk-spring system. $\theta \ll 1$. Find the equation of motion if $M=1kg$, $k=2 N/m$, $R=10cm$, $r=5cm$, $\alpha=8Ns/m$.



Using the Lagrangian:

$$T = E_c = T[\text{Disque}] = \frac{1}{2}J\dot{\theta}^2$$

$$J = \frac{1}{2}MR^2$$

$$T = \frac{1}{2}\left(\frac{1}{2}MR^2\right)\dot{\theta}^2$$

$$dU_k = -\vec{F}_k d\vec{l} = -F_k dl \cos(\vec{F}_k(\uparrow), d\vec{l}(\downarrow))$$

$$dU_k = -F_k dl \cos(180^\circ)$$

$$\cos(180^\circ) = -1$$

$$dU_k = +F_k dl = kx dx / F_k = kx, dl = dx$$

$$U_k = \int dU_k = \int_0^{x_0+x} kx dx = k \int_0^{x_0+x} x dx = k \left[\frac{x^2}{2} \right]_0^{x_0+x} = \frac{k}{2} [x^2]_0^{x_0+x}$$

$$U = U_k = \frac{k}{2} [(x_0 + x)^2 - 0]$$

$$U = \frac{k}{2} (x_0 + x)^2 = \frac{k}{2} (x^2 + 2xx_0 + x_0^2) = \frac{k}{2} x^2 + kxx_0 + \frac{k}{2} x_0^2$$

$$U = \frac{k}{2} x^2 + kxx_0 + \frac{k}{2} x_0^2 + cte / \frac{k}{2} x_0^2 = cte$$

Alors

$$U = U_k = \frac{k}{2} x^2 + kxx_0 + cte$$

Whith

$$\sin \theta = \frac{x}{R} \Rightarrow x = R \sin \theta$$

$$U = U_k = \frac{k}{2} x^2 + kxx_0 + cte = \frac{k}{2} (R \sin \theta)^2 + k(R \sin \theta)x_0 + cte$$

At low amplitude $\sin \theta \approx \theta$

$$U = \frac{k}{2} R^2 (\theta^2) + (kRx_0)(\theta) + cte$$

In the equilibrium

$$\left. \frac{\partial U}{\partial \theta} \right|_{\theta=0} = 0$$

$$\frac{\partial U}{\partial \theta} = \frac{k}{2} \times R^2 \times 2\theta - (kRx_0)$$

$$\left. \frac{\partial U}{\partial \theta} \right|_{\theta=0} : \theta = 0 \Rightarrow \frac{k}{2} \times R^2 \times 2(\theta = 0) - (kRx_0) = -(kRx_0)$$

$$\left. \frac{\partial U}{\partial x} \right|_{x=0} = 0 \Rightarrow -(kRx_0) = 0 \Rightarrow x_0 = 0$$

So

$$U = \frac{k}{2} R^2 \theta^2 + cte.$$

$$D = \frac{1}{2} \alpha v^2 = \frac{1}{2} \alpha \dot{x}^2$$

$$\dot{x} = r\dot{\theta} \Rightarrow D = \frac{1}{2}\alpha(r\dot{\theta})^2$$

$$\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{\theta}}\right) - \left(\frac{\partial L}{\partial \theta}\right) = -\frac{\partial D}{\partial \dot{\theta}}$$

$$\text{Avec } L = T - U = U = \frac{1}{2}\left(\frac{1}{2}MR^2\right)\dot{\theta}^2 - \frac{k}{2}R^2\theta^2 + \text{cte}$$

$$\frac{\partial L}{\partial \dot{\theta}} = \frac{\partial}{\partial \dot{\theta}}\left(\frac{1}{2}\left(\frac{1}{2}MR^2\right)\dot{\theta}^2\right) = \frac{1}{2}\left(\frac{1}{2}MR^2\right)\frac{\partial}{\partial \dot{\theta}}(\dot{\theta}^2) = \frac{1}{2}\left(\frac{1}{2}MR^2\right)(2\dot{\theta}) = \left(\frac{1}{2}MR^2\right)\dot{\theta}$$

$$\Rightarrow \frac{d}{dt}\left(\frac{\partial L}{\partial \dot{\theta}}\right) = \frac{d}{dt}\left(\left(\frac{1}{2}MR^2\right)\dot{\theta}\right) = \left(\frac{1}{2}MR^2\right)\frac{d}{dt}(\dot{\theta}) = \left(\frac{1}{2}MR^2\right)\ddot{\theta}$$

$$\frac{\partial L}{\partial \theta} = \frac{\partial}{\partial \theta}\left(-\frac{1}{2}kR^2\theta^2\right) = -\frac{1}{2}kR^2\frac{\partial}{\partial \theta}(\theta^2) = -\frac{1}{2}kR^2(2\theta) = -kR^2\theta$$

$$D = \frac{1}{2}\alpha v^2 = \frac{1}{2}\alpha r^2\dot{\theta}^2 \Rightarrow \frac{\partial D}{\partial \dot{\theta}} = \frac{\partial}{\partial \dot{\theta}}\left(\frac{1}{2}\alpha r^2\dot{\theta}^2\right) = \frac{1}{2}\alpha r^2\frac{\partial}{\partial \dot{\theta}}(\dot{\theta}^2) = \frac{1}{2}\alpha r^2(2\dot{\theta}) = \alpha r^2\dot{\theta}$$

So

$$\left(\frac{1}{2}MR^2\right)\ddot{\theta} + kR^2\theta = -\alpha r^2\dot{\theta} \Rightarrow \begin{cases} \ddot{\theta} + \frac{kR^2}{\left(\frac{1}{2}MR^2\right)}\theta + \frac{\alpha r^2}{\left(\frac{1}{2}MR^2\right)}\dot{\theta} = 0 \\ \ddot{\theta} + \omega_0^2\theta + 2\delta\dot{\theta} = 0 \end{cases}$$

$$\Rightarrow \begin{cases} \ddot{\theta} + \frac{2k}{M}\theta + \frac{2\alpha r^2}{MR^2}\dot{\theta} = 0 \\ \ddot{\theta} + \omega_0^2\theta + 2\delta\dot{\theta} = 0 \end{cases}$$

$$\omega_0^2 = \frac{2k}{M} \Rightarrow \omega_0 = \sqrt{\frac{2k}{M}} \text{ et } 2\delta = \frac{2\alpha r^2}{MR^2} \Rightarrow \delta = \frac{\alpha r^2}{MR^2}$$