

Chapter 5: Linear systems with two degrees of freedom

5.1 Systems with two degrees of freedom:

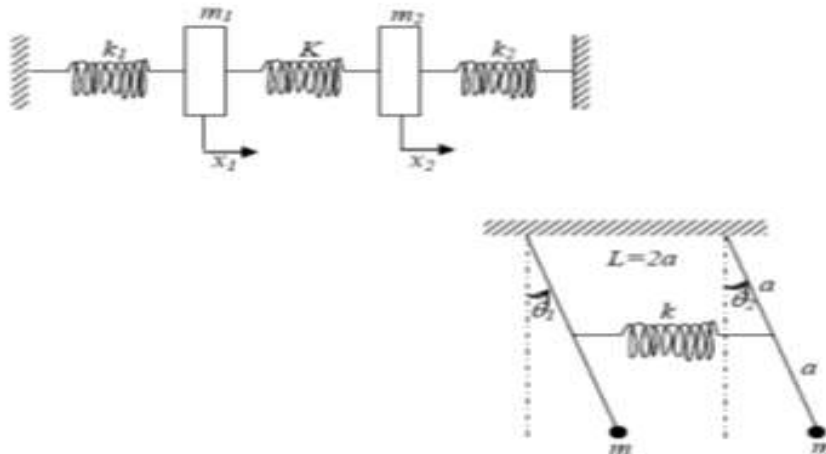
A system with two degrees of freedom can be considered as the coupling of two systems with a single degree of freedom. Therefore, we need two differential equations obtained from a system of two Lagrange equations

$$\begin{cases} \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_1} \right) + \frac{\partial D_a}{\partial \dot{q}_1} - \frac{\partial L}{\partial q_1} = F_{q_1} \\ \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_2} \right) + \frac{\partial D_a}{\partial \dot{q}_2} - \frac{\partial L}{\partial q_2} = F_{q_2} \end{cases}$$

5.2 Types of coupling:

a- Elastic coupling:

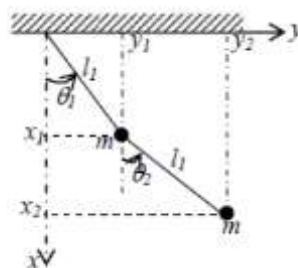
The coupling between the two systems is through a spring.



Elastic coupling

b- Inertial coupling:

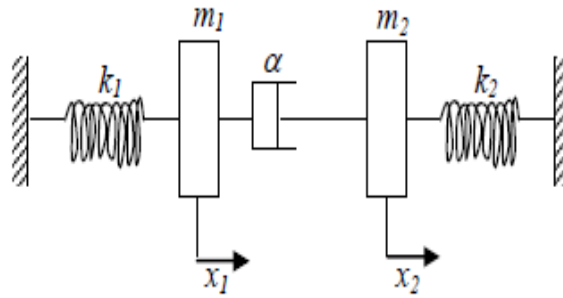
The coupling between the two systems is through a mass.



Inertial coupling

c-Viscous coupling:

The coupling between the two systems is through a damper.



Viscous coupling

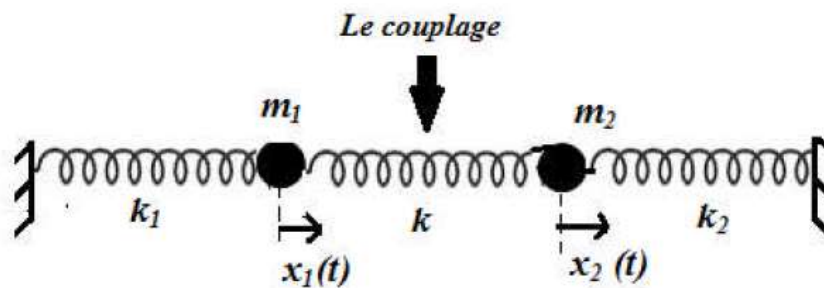
5.3 Free and undamped oscillation:

In the case of free and undamped oscillations, the Lagrange equations ($F_q = 0$ and $D_a = 0$) are reduced to:

$$\begin{cases} \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_1} \right) - \frac{\partial L}{\partial q_1} = 0 \\ \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_2} \right) - \frac{\partial L}{\partial q_2} = 0 \end{cases}$$

5.4 Equation of motion:

Consider the free opposite system. The two independent variables are x_1 and x_2 . k is called the coupling element.



- The kinetic energy is written as follows: $T = \frac{1}{2} m_1 \dot{x}_1^2 + \frac{1}{2} m_2 \dot{x}_2^2$
- The potential energy: $U = \frac{1}{2} k_1 x_1^2 + \frac{1}{2} k_2 x_2^2 + \frac{1}{2} k (x_1 - x_2)^2$
- Lagrange function is written:

$$L = T - U \Rightarrow L = \frac{1}{2} m_1 \dot{x}_1^2 + \frac{1}{2} m_2 \dot{x}_2^2 - \frac{1}{2} k_1 x_1^2 - \frac{1}{2} k_2 x_2^2 - \frac{1}{2} k (x_1 - x_2)^2$$
$$\Rightarrow L = \frac{1}{2} m_1 \dot{x}_1^2 + \frac{1}{2} m_2 \dot{x}_2^2 - \frac{1}{2} k_1 x_1^2 - \frac{1}{2} k_2 x_2^2 - \frac{1}{2} k (x_1^2 + x_2^2 - 2x_1 x_2)$$

$$\Rightarrow L = \frac{1}{2}m_1\dot{x}_1^2 + \frac{1}{2}m_2\dot{x}_2^2 - \frac{1}{2}k_1x_1^2 - \frac{1}{2}k_2x_2^2 - \frac{1}{2}kx_1^2 - \frac{1}{2}kx_2^2 - \frac{1}{2}k(-2x_1x_2)$$

$$\Rightarrow L = \frac{1}{2}m_1\dot{x}_1^2 + \frac{1}{2}m_2\dot{x}_2^2 - \frac{1}{2}k_1x_1^2 - \frac{1}{2}k_2x_2^2 - \frac{1}{2}kx_1^2 - \frac{1}{2}kx_2^2 + kx_1x_2$$

- The two Lagrange equations are written: (For $D_a = 0, F = 0$:
system non damped and non forced)

$$\begin{cases} \frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}_1}\right) - \left(\frac{\partial L}{\partial x_1}\right) = 0 \\ \frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}_2}\right) - \left(\frac{\partial L}{\partial x_2}\right) = 0 \end{cases}$$

The coupled differential system then becomes:

$$\begin{cases} \frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}_1}\right) - \left(\frac{\partial L}{\partial x_1}\right) = 0 \\ \frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}_2}\right) - \left(\frac{\partial L}{\partial x_2}\right) = 0 \end{cases} \Rightarrow \begin{cases} \frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}_1}\right) = m_1\ddot{x}_1, \left(\frac{\partial L}{\partial x_1}\right) = -k_1x_1 - kx_1 + kx_2 \\ \frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}_2}\right) = m_2\ddot{x}_2, \left(\frac{\partial L}{\partial x_2}\right) = -k_2x_2 - kx_2 + kx_1 \end{cases}$$

$$\Rightarrow \begin{cases} m_1\ddot{x}_1 + (k_1 + k)x_1 - kx_2 = 0 \\ m_2\ddot{x}_2 + (k + k_2)x_2 - kx_1 = 0 \end{cases} \Rightarrow \begin{cases} \ddot{x}_1 + \frac{(k_1 + k)}{m_1}x_1 - \frac{k}{m_1}x_2 = 0 \\ \ddot{x}_2 + \frac{(k + k_2)}{m_2}x_2 - \frac{k}{m_2}x_1 = 0 \end{cases}$$

5.5 the propre modes:

For the system of two differential equations to admit a solution, it is necessary that $x_1(t)$ and $x_2(t)$ are two harmonic functions of the same pulsation ω , but of different amplitudes A_1 and A_2 :

$$x_1(t) = A_1 \cos(\omega t + \varphi) \Rightarrow \ddot{x}_1(t) = -A_1 \omega^2 \cos(\omega t + \varphi)$$

$$x_2(t) = A_2 \cos(\omega t + \varphi) \Rightarrow \ddot{x}_2(t) = -A_2 \omega^2 \cos(\omega t + \varphi)$$

To find ω , use the complex representation:

$$\Rightarrow \begin{cases} x_1 = A_1 \cos(\omega t + \varphi_1) \rightarrow \overline{x_1} = A_1 e^{j(\omega t + \varphi_1)} = \overline{A_1} e^{j\omega t} \\ x_2 = A_2 \cos(\omega t + \varphi_2) \rightarrow \overline{x_2} = A_2 e^{j(\omega t + \varphi_2)} = \overline{A_2} e^{j\omega t} \end{cases}$$

$$\Rightarrow \begin{cases} \overline{\dot{x}_1} = \overline{A_1} e^{j\omega t} \\ \overline{\dot{x}_2} = \overline{A_2} e^{j\omega t} \end{cases} \Rightarrow \begin{cases} \overline{\ddot{x}_1} = j\omega \overline{A_1} e^{j\omega t} \\ \overline{\ddot{x}_2} = j\omega \overline{A_2} e^{j\omega t} \end{cases} \Rightarrow \begin{cases} \overline{\ddot{x}_1} = (j\omega)^2 \overline{A_1} e^{j\omega t} = -\omega^2 \overline{A_1} e^{j\omega t} \\ \overline{\ddot{x}_2} = (j\omega)^2 \overline{A_2} e^{j\omega t} = -\omega^2 \overline{A_2} e^{j\omega t} \end{cases}$$

$$\Rightarrow \begin{cases} \ddot{x}_1 + \frac{(k_1 + k)}{m_1}x_1 - \frac{k}{m_1}x_2 = 0 \\ \ddot{x}_2 + \frac{(k + k_2)}{m_2}x_2 - \frac{k}{m_2}x_1 = 0 \end{cases} \Rightarrow \begin{cases} -\omega^2 \overline{A_1} e^{j\omega t} + \frac{(k_1 + k)}{m_1} \overline{A_1} e^{j\omega t} - \frac{k}{m_1} \overline{A_2} e^{j\omega t} = 0 \\ -\omega^2 \overline{A_2} e^{j\omega t} + \frac{(k + k_2)}{m_2} \overline{A_2} e^{j\omega t} - \frac{k}{m_2} \overline{A_1} e^{j\omega t} = 0 \end{cases}$$

$$\begin{cases} -\omega^2 \bar{A}_1 e^{j\omega t} + \frac{(k_1 + k)}{m_1} \bar{A}_1 e^{j\omega t} - \frac{k}{m_1} \bar{A}_2 e^{j\omega t} = 0 \\ -\omega^2 \bar{A}_2 e^{j\omega t} + \frac{(k + k_2)}{m_2} \bar{A}_2 e^{j\omega t} - \frac{k}{m_2} \bar{A}_1 e^{j\omega t} = 0 \end{cases} \Rightarrow \begin{cases} \left(-\omega^2 + \frac{(k_1 + k)}{m_1} \right) \bar{A}_1 - \left(\frac{k}{m_1} \right) \bar{A}_2 = 0 \\ \left(-\omega^2 + \frac{(k + k_2)}{m_2} \right) \bar{A}_2 - \left(\frac{k}{m_2} \right) \bar{A}_1 = 0 \end{cases}$$

$$\Rightarrow \begin{cases} \left(-\omega^2 + \frac{(k_1 + k)}{m_1} \right) \bar{A}_1 - \left(\frac{k}{m_1} \right) \bar{A}_2 = 0 \\ -\left(\frac{k}{m_2} \right) \bar{A}_1 + \left(-\omega^2 + \frac{(k + k_2)}{m_2} \right) \bar{A}_2 = 0 \end{cases} \Rightarrow \begin{cases} (-\omega^2 + a) \bar{A}_1 - b \bar{A}_2 = 0 \\ -c \bar{A}_1 + (-\omega^2 + d) \bar{A}_2 = 0 \end{cases}$$

$$a = \frac{k + k_1}{m_1}, \quad b = \frac{k}{m_1}, \quad c = \frac{k}{m_2}, \quad d = \frac{k + k_2}{m_2}.$$

We can write in matrix form as follows:

$$\begin{pmatrix} (a - \omega^2) & -b \\ -c & (d - \omega^2) \end{pmatrix} \begin{pmatrix} \bar{A}_1 \\ \bar{A}_2 \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}$$

For the equation to be true without $\bar{A}_1$ and $\bar{A}_2$ both being zero, its characteristic determinant must be zero:

$$\det = \begin{vmatrix} (a - \omega^2) & -b \\ -c & (d - \omega^2) \end{vmatrix} = (a - \omega^2)(d - \omega^2) - (-c)(-b) = 0$$

$$\Rightarrow \det = \omega^4 - \omega^2 d - \omega^2 a + ad - bc = \omega^4 - (d + a)\omega^2 + (ad - bc) = 0$$

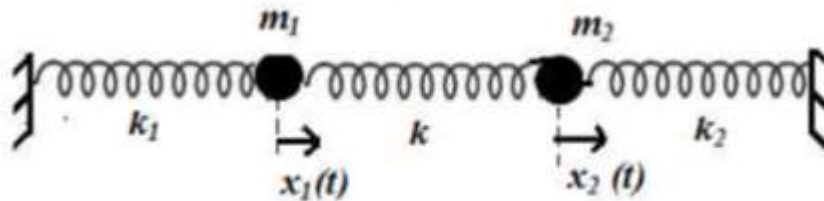
This gives us the characteristic equation: $\omega^4 - (d + a)\omega^2 + (ad - bc) = 0$

The two real and positive solutions ω_1 and ω_2 of this equation are called proper or normal pulsations. The smaller one is called the fundamental, the other is called the harmonic.

- *First propre mode*: For $\omega = \omega_1$, the system implies that: $\frac{A_{1(1^{st} \text{ mode})}}{A_{2(1^{st} \text{ mode})}} = \frac{d - \omega_1^2}{c} > 0$

The solution is written in this case:

$$\begin{cases} x_{1(1^{st} \text{ mode})}(t) = A_{1(1^{st} \text{ mode})} \cos(\omega_1 t + \varphi) \\ x_{2(1^{st} \text{ mode})}(t) = A_{2(1^{st} \text{ mode})} \cos(\omega_1 t + \varphi) \end{cases}$$

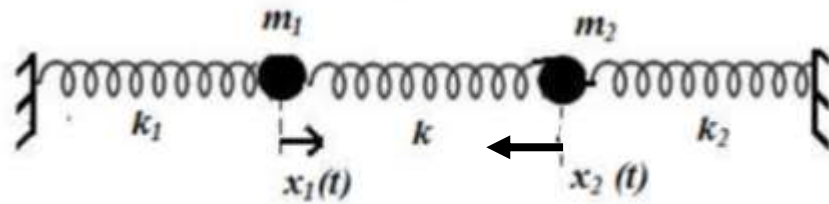


1st propre mode

- *Second propre mode*: For $\omega=\omega_2$, the system implies that: $\frac{A_{1(2^{nd} \text{ mode})}}{A_{2(2^{nd} \text{ mode})}} = \frac{d-\omega_1^2}{c} < 0$

The solution is written in this case:

$$\begin{cases} x_{1(2^{nd} \text{ mode})}(t) = A_{1(2^{nd} \text{ mode})} \cos(\omega_1 t + \varphi) \\ x_{2(2^{nd} \text{ mode})}(t) = -A_{2(2^{nd} \text{ mode})} \cos(\omega_1 t + \varphi) \end{cases}$$



2nd propre mode