

## Recapitulative Récapitulatif

### 1. Approximation of displacement fields of some elements:

(Approximation des champs de déplacements de quelques éléments)

#### 1.1. Two-nodes tension-compression bar element:

(Elément barre de traction-compression à deux nœuds)

$$U = u \quad \text{et} \quad \{d\} = \begin{Bmatrix} d_1 \\ d_2 \end{Bmatrix} = \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix},$$

$$u = [N] \times \{d\} = [N_1 \quad N_2] \{d\} = \left[ 1 - \frac{x}{L} \quad \frac{x}{L} \right] \times \{d\}$$

#### 1.2. Two-nodes torsion bar element:

(Elément barre de torsion à deux nœuds)

$$U = \theta_x \quad \text{et} \quad \{d\} = \begin{Bmatrix} d_1 \\ d_2 \end{Bmatrix} = \begin{Bmatrix} \theta_{x1} \\ \theta_{x2} \end{Bmatrix},$$

$$\theta_x = [N] \times \{d\} = [N_1 \quad N_2] \{d\} = \left[ 1 - \frac{x}{L} \quad \frac{x}{L} \right] \times \{d\}$$

#### 1.3 Two-nodes bending bar element (Beam element in the x-y plane)

Elément barre de flexion à deux nœuds (Elément poutre dans le plan x-y)

##### 1.3.1 Standard element: (Elément standard)

$$U = \begin{Bmatrix} v \\ \theta \end{Bmatrix} \quad \text{et} \quad \{d\} = \begin{Bmatrix} d_1 \\ d_2 \\ d_3 \\ d_4 \end{Bmatrix} = \begin{Bmatrix} v_1 \\ \theta_1 \\ v_2 \\ \theta_2 \end{Bmatrix} \quad \text{with} \quad \theta = \theta_z = \frac{dv}{dx}$$

$$v = [N] \times \{d\} = \left[ 1 - \frac{3x^2}{L^2} + \frac{2x^3}{L^3} \quad x - \frac{2x^2}{L} + \frac{x^3}{L^2} \quad \frac{3x^2}{L^2} - \frac{2x^3}{L^3} \quad -\frac{x^2}{L} + \frac{x^3}{L^2} \right] \times \{d\}$$

$$\theta = \frac{dv}{dx} = [N'] \times \{d\} = \left[ -\frac{6x}{L^2} + \frac{6x^2}{L^3} \quad 1 - \frac{4x}{L} + \frac{3x^2}{L^2} \quad \frac{6x}{L^2} - \frac{6x^2}{L^3} \quad -\frac{2x}{L} + \frac{3x^2}{L^2} \right] \times \{d\}$$

##### 1.3.2 Left hinged beam element : (Elément articulée à gauche)

$$v = [N] \times \{d\} = \left[ 1 - \frac{3x}{2L} + \frac{x^3}{2L^3} \quad 0 \quad \frac{3x}{2L} - \frac{x^3}{2L^3} \quad -\frac{x}{2} + \frac{x^3}{2L^2} \right] \times \{d\}$$

$$\theta = \frac{dv}{dx} = [N'] \times \{d\} = \left[ -\frac{3}{2L} + \frac{3x^2}{2L^3} \quad 0 \quad \frac{3}{2L} - \frac{3x^2}{2L^3} \quad -\frac{1}{2} + \frac{3x^2}{2L^2} \right] \times \{d\}$$

### 1.3.3 Right hinged beam element: (Elément articulé à droite)

$$v = [N] \times \{d\} = \begin{bmatrix} 1 - \frac{3x^2}{2L^2} + \frac{x^3}{2L^3} & x - \frac{3x^2}{2L} + \frac{x^3}{2L^2} & \frac{3x^2}{2L^2} - \frac{x^3}{2L^3} & 0 \end{bmatrix} \times \{d\}$$

$$\theta = \frac{dv}{dx} = [N'] \times \{d\} = \begin{bmatrix} -\frac{3x}{L^2} + \frac{3x^2}{2L^3} & 1 - \frac{3x}{L} + \frac{3x^2}{2L^2} & \frac{3x}{L^2} - \frac{3x^2}{2L^3} & 0 \end{bmatrix} \times \{d\}$$

### 1.3.4 Hinged beam element at its two nodes: (Elément articulé à ses deux nœuds)

$$v = [N] \times \{d\} = \begin{bmatrix} 1 - \frac{x}{L} & 0 & \frac{x}{L} & 0 \end{bmatrix} \times \{d\}$$

$$\theta = \frac{dv}{dx} = [N'] \times \{d\} = \begin{bmatrix} -\frac{1}{L} & 0 & \frac{1}{L} & 0 \end{bmatrix} \times \{d\}$$

### 1.4 Two-nodes tension–compression and bending bar element (2D frame element in the x-y plane): Elément barre de traction –compression et de flexion à deux nœuds (Elément portique 2D dans le plan x-y)

$$U = \begin{Bmatrix} u \\ v \\ \theta \end{Bmatrix} \text{ et } \{d\} = \begin{Bmatrix} d_1 \\ d_2 \\ d_3 \\ d_4 \\ d_5 \\ d_6 \end{Bmatrix} = \begin{Bmatrix} u_1 \\ v_1 \\ \theta_1 \\ u_2 \\ v_2 \\ \theta_2 \end{Bmatrix}, \quad \theta = \theta_z$$

#### 1.4.1 Standard 2D Frame element:

$$u = \begin{bmatrix} 1 - \frac{x}{L} & 0 & 0 & \frac{x}{L} & 0 & 0 \end{bmatrix} \times \{d\}$$

$$w = \begin{bmatrix} 0 & 1 - \frac{3x^2}{L^2} + \frac{2x^3}{L^3} & x - \frac{2x^2}{L} + \frac{x^3}{L^2} & 0 & \frac{3x^2}{L^2} - \frac{2x^3}{L^3} & -\frac{x^2}{L} + \frac{x^3}{L^2} \end{bmatrix} \times \{d\}$$

$$\theta = -\frac{dw}{dx} = \begin{bmatrix} 0 & -\frac{6x}{L^2} + \frac{6x^2}{L^3} & 1 - \frac{4x}{L} + \frac{3x^2}{L^2} & 0 & \frac{6x}{L^2} - \frac{6x^2}{L^3} & -\frac{2x}{L} + \frac{3x^2}{L^2} \end{bmatrix} \times \{d\}$$

#### 1.4.2 Left hinged 2D Frame element:

$$u = \begin{bmatrix} 1 - \frac{x}{L} & 0 & 0 & \frac{x}{L} & 0 & 0 \end{bmatrix} \times \{d\}$$

$$v = [N] \times \{d\} = \begin{bmatrix} 0 & 1 - \frac{3x}{2L} + \frac{x^3}{2L^3} & 0 & 0 & \frac{3x}{2L} - \frac{x^3}{2L^3} & -\frac{x}{2} + \frac{x^3}{2L^2} \end{bmatrix} \times \{d\}$$

$$\theta = \frac{dv}{dx} = [N'] \times \{d\} = \begin{bmatrix} 0 & -\frac{3}{2L} + \frac{3x^2}{2L^3} & 0 & 0 & \frac{3}{2L} - \frac{3x^2}{2L^3} & -\frac{1}{2} + \frac{3x^2}{2L^2} \end{bmatrix} \times \{d\}$$

#### 1.4.3 Right hinged 2D Frame element:

$$u = \begin{bmatrix} 1 - \frac{x}{L} & 0 & 0 & \frac{x}{L} & 0 & 0 \end{bmatrix} \times \{d\}$$

$$v = [N] \times \{d\} = \begin{bmatrix} 0 & 1 - \frac{3x^2}{2L^2} + \frac{x^3}{2L^3} & x - \frac{3x^2}{2L} + \frac{x^3}{2L^2} & 0 & \frac{3x^2}{2L^2} - \frac{x^3}{2L^3} & 0 \end{bmatrix} \times \{d\}$$

$$\theta = \frac{dv}{dx} = [N'] \times \{d\} = \begin{bmatrix} 0 & -\frac{3x}{L^2} + \frac{3x^2}{2L^3} & 1 - \frac{3x}{L} + \frac{3x^2}{2L^2} & 0 & \frac{3x}{L^2} - \frac{3x^2}{2L^3} & 0 \end{bmatrix} \times \{d\}$$

#### 1.4.4 Hinged 2D Frame element at its two nodes:

$$u = \begin{bmatrix} 1 - \frac{x}{L} & 0 & 0 & \frac{x}{L} & 0 & 0 \end{bmatrix} \times \{d\}$$

$$v = [N] \times \{d\} = \begin{bmatrix} 0 & 1 - \frac{x}{L} & 0 & 0 & \frac{x}{L} & 0 \end{bmatrix} \times \{d\}$$

$$\theta = \frac{dv}{dx} = [N'] \times \{d\} = \begin{bmatrix} 0 & -\frac{1}{L} & 0 & 0 & \frac{1}{L} & 0 \end{bmatrix} \times \{d\}$$

#### 1.5. Bending and Torsion Bar Element (crossed beam element)

$$U = \begin{Bmatrix} w \\ \theta_x \\ \theta_y \end{Bmatrix}, \quad \{d\} = \begin{Bmatrix} d_1 \\ d_2 \\ d_3 \\ d_4 \\ d_5 \\ d_6 \end{Bmatrix} = \begin{Bmatrix} w_1 \\ \theta_{x1} \\ \theta_{y1} \\ w_2 \\ \theta_{x2} \\ \theta_{y2} \end{Bmatrix},$$

##### Standard Crossed Beam element:

$$w = \begin{bmatrix} 1 - \frac{3x^2}{L^2} + \frac{2x^3}{L^3} & 0 & -x + \frac{2x^2}{L} - \frac{x^3}{L^2} & \frac{3x^2}{L^2} - \frac{2x^3}{L^3} & 0 & \frac{x^2}{L} - \frac{x^3}{L^2} \end{bmatrix} \times \{d\}$$

$$\theta_x = \begin{bmatrix} 0 & 1 - \frac{x}{L} & 0 & 0 & \frac{x}{L} & 0 \end{bmatrix} \times \{d\}$$

$$\theta_y = -\frac{dw}{dx} = \begin{bmatrix} \frac{6x}{L^2} - \frac{6x^2}{L^3} & 0 & 1 - \frac{4x}{L} + \frac{3x^2}{L^2} & \frac{6x}{L^2} - \frac{6x^2}{L^3} & 0 & -\frac{2x}{L} + \frac{3x^2}{L^2} \end{bmatrix} \times \{d\}$$

#### 1.6. Three nodes Plane Elasticity Triangular element (Élément triangulaire d'élasticité plane à trois nœuds)

$$U = \begin{Bmatrix} u \\ v \end{Bmatrix}, \{d\} = \begin{Bmatrix} d_1 \\ d_2 \\ d_3 \\ d_4 \\ d_5 \\ d_6 \end{Bmatrix} = \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \end{Bmatrix}$$

$$\begin{Bmatrix} u \\ v \end{Bmatrix} = \begin{bmatrix} N_1 & 0 & N_2 & 0 & N_3 & 0 \\ 0 & N_1 & 0 & N_2 & 0 & N_3 \end{bmatrix} \begin{Bmatrix} d_1 \\ d_2 \\ d_3 \\ d_4 \\ d_5 \\ d_6 \end{Bmatrix}$$

$$N_1 = \frac{1}{2\Delta} [(x_2 y_3 - y_2 x_3) + x(y_2 - y_3) + y(x_3 - x_2)]$$

$$N_2 = \frac{1}{2\Delta} [(y_1 x_3 - x_1 y_3) + x(y_3 - y_1) + y(x_1 - x_3)]$$

$$N_3 = \frac{1}{2\Delta} [(x_1 y_2 - y_1 x_2) + x(y_1 - y_2) + y(x_2 - x_1)]$$

$\Delta$  is the area of the triangular element and  $\{x_i, y_i\}$  are the coordinates of its nodes

$$2\Delta = \begin{vmatrix} 1 & x_1 & y_1 \\ 1 & x_2 & y_2 \\ 1 & x_3 & y_3 \end{vmatrix}$$

### 1.7 Tetrahedral element (Volumic four nodes element):

$$U = \begin{Bmatrix} u \\ v \\ w \end{Bmatrix}, \{d\} = [d_1 \ d_2 \ d_3 \ d_4 \ d_5 \ d_6 \ d_7 \ d_8 \ d_9 \ d_{10} \ d_{11} \ d_{12}]^T$$

$$\{d\} = [u_1 \ v_1 \ w_1 \ u_2 \ v_2 \ w_2 \ u_3 \ v_3 \ w_3 \ u_4 \ v_4 \ w_4]^T$$

$$U = \begin{Bmatrix} u \\ v \\ w \end{Bmatrix} = \begin{bmatrix} N_1 & 0 & 0 & N_2 & 0 & 0 & N_3 & 0 & 0 & N_4 & 0 & 0 \\ 0 & N_1 & 0 & 0 & N_2 & 0 & 0 & N_3 & 0 & 0 & N_4 & 0 \\ 0 & 0 & N_1 & 0 & 0 & N_2 & 0 & 0 & N_3 & 0 & 0 & N_4 \end{bmatrix} \{d\}$$

$$N_i = L_i = \frac{1}{6V} (a_i + b_i x + c_i y + d_i z)$$

Where:

$V$  is the volume of the tetrahedral element

$$6V = \det \begin{bmatrix} 1 & 1 & 1 & 1 \\ x_1 & x_2 & x_3 & x_4 \\ y_1 & y_2 & y_3 & y_4 \\ z_1 & z_2 & z_3 & z_4 \end{bmatrix}$$

$$a_i = c_{1i}^T ; b_i = c_{2i}^T ; c_i = c_{3i}^T ; d_i = c_{4i}^T$$

$$c_{11} = \det \begin{bmatrix} x_2 & x_3 & x_4 \\ y_2 & y_3 & y_4 \\ z_2 & z_3 & z_4 \end{bmatrix}; c_{12} = \det \begin{bmatrix} x_1 & x_3 & x_4 \\ y_1 & y_3 & y_4 \\ z_1 & z_3 & z_4 \end{bmatrix}; c_{13} = \det \begin{bmatrix} x_1 & x_2 & x_4 \\ y_1 & y_2 & y_4 \\ z_1 & z_2 & z_4 \end{bmatrix}$$

$$c_{14} = \det \begin{bmatrix} x_1 & x_2 & x_3 \\ y_1 & y_2 & y_3 \\ z_1 & z_2 & z_3 \end{bmatrix}$$

## 2 Stiffness matrices of some elements: (Matrices de rigidités de quelques éléments)

Let  $[T]$ ,  $[k]$ , and  $[K]$  denote respectively the transformation matrix, the stiffness matrix in the local frame, and the stiffness matrix in the global frame. ((Notons respectivement par  $[T]$ ,  $[k]$  et  $[K]$  la matrice de transformation, la matrice de rigidité dans le repère local et la matrice de rigidité dans le repère global)

### 2.1 Spring element : Élément ressort :

$$T = \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix} \quad [k] = k \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \quad [K] = k \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

### 2.2 Two-node tension–compression bar element:

(Élément barre de traction compression à deux nœuds)

$$T = \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix} \quad [k] = \frac{EA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \quad [K] = \frac{EA}{L} \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

### 2.3. Bending bar element (Beam element) (Élément barre de flexion (Élément poutre))

in the  $x$ - $y$  plane:  $I = I_z$ ,  $\theta = \theta_z$

#### 2.3.1 Standard Beam element:

$$[k] = [K] = \frac{EI}{L^3} \begin{bmatrix} 12 & 6L & -12 & 6L \\ 6L & 4L^2 & -6L & 2L^2 \\ -12 & -6L & 12 & -6L \\ 6L & 2L^2 & -6L & 4L^2 \end{bmatrix}$$

### 2.3.2 Left hinged Beam element:

$$\theta_1 = -\frac{3}{2L}v_1 + \frac{3}{2L}v_2 - \frac{1}{2}\theta_2$$

$$[k] = \frac{3EI_z}{L^3} \begin{bmatrix} 1 & 0 & -1 & L \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & -L \\ L & 0 & -L & L^2 \end{bmatrix}$$

### 2.3.3 Right hinged Beam element:

$$\theta_2 = -\frac{3}{2L}v_1 + \frac{3}{2L}v_2 - \frac{1}{2}\theta_1$$

$$[k] = \frac{3EI_z}{L^3} \begin{bmatrix} 1 & L & -1 & 0 \\ L & L^2 & -L & 0 \\ -1 & -L & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

### 2.3.4 Beam element hinged at its two nodes:

$$\theta_1 = \frac{-v_1}{L} + \frac{v_2}{L} = \theta_2$$

$$[k] = \frac{3EI_z}{L^3} \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

## 2.4 Bending and tension compression barre element (2D frame element) (Elément barre se flexion et traction compression (Elément portique 2D))

In the x-y local coordinates:  $I = I_z$ ,  $\theta = \theta_z$

$$T = \begin{bmatrix} l & m & 0 & 0 & 0 & 0 \\ -m & l & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & l & m & 0 \\ 0 & 0 & 0 & -m & l & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

#### 2.4.1 Standard 2D Frame element:

$$[k] = \frac{E}{L} \begin{bmatrix} A & 0 & 0 & -A & 0 & 0 \\ 0 & \frac{12I}{L^2} & \frac{6I}{L} & 0 & \frac{-12I}{L^2} & \frac{6I}{L} \\ 0 & \frac{6I}{L} & 4I & 0 & \frac{-6I}{L} & 2I \\ -A & 0 & 0 & A & 0 & 0 \\ 0 & \frac{-12I}{L^2} & \frac{-6I}{L} & 0 & \frac{12I}{L^2} & \frac{-6I}{L} \\ 0 & \frac{6I}{L} & 2I & 0 & \frac{-6I}{L} & 4I \end{bmatrix}$$

$$[K] = \frac{E}{L} \begin{bmatrix} +\left(Al^2 + \frac{12I}{L^2}m^2\right) & +lm\left(A - \frac{12I}{L^2}\right) & -\frac{6ml}{L} & -\left(Al^2 + \frac{12I}{L^2}m^2\right) & -lm\left(A - \frac{12I}{L^2}\right) & -\frac{6ml}{L} \\ +lm\left(A - \frac{12I}{L^2}\right) & +\left(Am^2 + \frac{12I}{L^2}l^2\right) & \frac{6Il}{L} & -lm\left(A - \frac{12I}{L^2}\right) & -\left(Am^2 + \frac{12I}{L^2}l^2\right) & \frac{6Il}{L} \\ -\frac{6ml}{L} & \frac{6Il}{L} & 4I & \frac{6ml}{L} & -\frac{6Il}{L} & 2I \\ -\left(Al^2 + \frac{12I}{L^2}m^2\right) & -lm\left(A - \frac{12I}{L^2}\right) & \frac{6ml}{L} & +\left(Al^2 + \frac{12I}{L^2}m^2\right) & +lm\left(A - \frac{12I}{L^2}\right) & \frac{6ml}{L} \\ -lm\left(A - \frac{12I}{L^2}\right) & -\left(Am^2 + \frac{12I}{L^2}l^2\right) & -\frac{6Il}{L} & +lm\left(A - \frac{12I}{L^2}\right) & +\left(Am^2 + \frac{12I}{L^2}l^2\right) & -\frac{6Il}{L} \\ -\frac{6ml}{L} & \frac{6Il}{L} & 2I & \frac{6ml}{L} & -\frac{6Il}{L} & 4I \end{bmatrix}$$

#### 2.4.2 Left hinged 2D Frame element

$$[k] = \frac{E}{L} \begin{bmatrix} A & 0 & 0 & -A & 0 & 0 \\ 0 & \frac{3I}{L^2} & 0 & 0 & \frac{3I}{L^2} & \frac{3I}{L} \\ 0 & 0 & 0 & 0 & 0 & 0 \\ -A & 0 & 0 & A & 0 & 0 \\ 0 & \frac{-3I}{L^2} & 0 & 0 & \frac{3I}{L^2} & \frac{-3I}{L} \\ 0 & \frac{3I}{L} & 0 & 0 & \frac{-3I}{L} & 3I \end{bmatrix}$$

$$[K] = \frac{E}{L} \begin{bmatrix} +\left(Al^2 + \frac{3I}{L^2}m^2\right) & +lm\left(A - \frac{3I}{L^2}\right) & 0 & -\left(Al^2 + \frac{3I}{L^2}m^2\right) & -lm\left(A - \frac{3I}{L^2}\right) & -\frac{3ml}{L} \\ +lm\left(A - \frac{3I}{L^2}\right) & +\left(Am^2 + \frac{3I}{L^2}l^2\right) & 0 & -lm\left(A - \frac{3I}{L^2}\right) & -\left(Am^2 + \frac{3I}{L^2}l^2\right) & \frac{3Il}{L} \\ 0 & 0 & 0 & 0 & 0 & 0 \\ -\left(Al^2 + \frac{3I}{L^2}m^2\right) & -lm\left(A - \frac{3I}{L^2}\right) & 0 & +\left(Al^2 + \frac{3I}{L^2}m^2\right) & +lm\left(A - \frac{3I}{L^2}\right) & \frac{3ml}{L} \\ -lm\left(A - \frac{3I}{L^2}\right) & -\left(Am^2 + \frac{3I}{L^2}l^2\right) & 0 & +lm\left(A - \frac{3I}{L^2}\right) & +\left(Am^2 + \frac{3I}{L^2}l^2\right) & -\frac{3Il}{L} \\ -\frac{3ml}{L} & \frac{3Il}{L} & 0 & \frac{3ml}{L} & -\frac{3Il}{L} & 3I \end{bmatrix}$$

### 2.4.3 Right hinged 2D Frame element

$$[k] = \frac{E}{L} \begin{bmatrix} A & 0 & 0 & -A & 0 & 0 \\ 0 & \frac{3I}{L^2} & \frac{3I}{L} & 0 & \frac{-3I}{L^2} & 0 \\ 0 & \frac{3I}{L} & 3I & 0 & \frac{-3I}{L} & 0 \\ -A & 0 & 0 & A & 0 & 0 \\ 0 & \frac{-3I}{L^2} & \frac{-3I}{L} & 0 & \frac{3I}{L^2} & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$[K] = \frac{E}{L} \begin{bmatrix} +\left(Al^2 + \frac{3I}{L^2}m^2\right) & +lm\left(A - \frac{3I}{L^2}\right) & -\frac{3ml}{L} & -\left(Al^2 + \frac{3I}{L^2}m^2\right) & -lm\left(A - \frac{3I}{L^2}\right) & 0 \\ +lm\left(A - \frac{3I}{L^2}\right) & +\left(Am^2 + \frac{3I}{L^2}l^2\right) & \frac{3Il}{L} & -lm\left(A - \frac{3I}{L^2}\right) & -\left(Am^2 + \frac{3I}{L^2}l^2\right) & 0 \\ -\frac{3ml}{L} & \frac{3Il}{L} & 3I & \frac{3ml}{L} & -\frac{3Il}{L} & 0 \\ -\left(Al^2 + \frac{3I}{L^2}m^2\right) & -lm\left(A - \frac{3I}{L^2}\right) & \frac{3ml}{L} & +\left(Al^2 + \frac{3I}{L^2}m^2\right) & +lm\left(A - \frac{3I}{L^2}\right) & 0 \\ -lm\left(A - \frac{3I}{L^2}\right) & -\left(Am^2 + \frac{3I}{L^2}l^2\right) & -\frac{3Il}{L} & +lm\left(A - \frac{3I}{L^2}\right) & +\left(Am^2 + \frac{3I}{L^2}l^2\right) & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

### 2.4.5 2D Frame element hinged at its two nodes

$$[k] = \frac{E}{L} \begin{bmatrix} A & 0 & 0 & -A & 0 & 0 \\ 0 & \frac{3I}{L^2} & 0 & 0 & \frac{-3I}{L^2} & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ -A & 0 & 0 & A & 0 & 0 \\ 0 & \frac{-3I}{L^2} & 0 & 0 & \frac{3I}{L^2} & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$[K] = \frac{E}{L} \begin{bmatrix} +\left(Al^2 + \frac{3I}{L^2}m^2\right) & +lm\left(A - \frac{3I}{L^2}\right) & 0 & -\left(Al^2 + \frac{3I}{L^2}m^2\right) & -lm\left(A - \frac{3I}{L^2}\right) & 0 \\ +lm\left(A - \frac{3I}{L^2}\right) & +\left(Am^2 + \frac{3I}{L^2}l^2\right) & 0 & -lm\left(A - \frac{3I}{L^2}\right) & -\left(Am^2 + \frac{3I}{L^2}l^2\right) & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ -\left(Al^2 + \frac{3I}{L^2}m^2\right) & -lm\left(A - \frac{3I}{L^2}\right) & 0 & +\left(Al^2 + \frac{3I}{L^2}m^2\right) & +lm\left(A - \frac{3I}{L^2}\right) & 0 \\ -lm\left(A - \frac{3I}{L^2}\right) & -\left(Am^2 + \frac{3I}{L^2}l^2\right) & 0 & +lm\left(A - \frac{3I}{L^2}\right) & +\left(Am^2 + \frac{3I}{L^2}l^2\right) & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

**2.5. Bending and Torsion Bar Element (crossed beam element):** (Élément barre de torsion et de flexion (Élément poutre croisée))

**Standard Crossed Beam element**

$$[T] = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & l & m & 0 & 0 & 0 \\ 0 & -m & l & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & l & m \\ 0 & 0 & 0 & 0 & -m & l \end{bmatrix}$$

$$[k] = \frac{E}{L} \begin{bmatrix} \frac{12I_y}{L^2} & 0 & \frac{-6I_y}{L} & \frac{-12I_y}{L^2} & 0 & \frac{-6I_y}{L} \\ 0 & \frac{J}{2(1+\nu)} & 0 & 0 & \frac{-J}{2(1+\nu)} & 0 \\ \frac{-6I_y}{L} & 0 & 4I_y & \frac{6I_y}{L} & 0 & 2I_y \\ \frac{-12I_y}{L^2} & 0 & \frac{6I_y}{L} & \frac{12I_y}{L^2} & 0 & \frac{6I_y}{L} \\ 0 & \frac{-J}{2(1+\nu)} & 0 & 0 & \frac{J}{2(1+\nu)} & 0 \\ \frac{-6I_y}{L} & 0 & 2I_y & \frac{6I_y}{L} & 0 & 4I_y \end{bmatrix}$$

$$[K] = \frac{E}{L} \begin{bmatrix} \frac{12I}{L^2} & \frac{6mI}{L} & \frac{-6lI}{L} & \frac{-12I}{L^2} & \frac{6mI}{L} & \frac{-6lI}{L} \\ \frac{6mI}{L} & \left( \frac{l^2J}{2(1+\nu)} + 4m^2I \right) & lm \left( \frac{J}{2(1+\nu)} - 4I \right) & \frac{-6mI}{L} & \left( \frac{-l^2J}{2(1+\nu)} + 2m^2I \right) & -lm \left( \frac{J}{2(1+\nu)} + 2I \right) \\ \frac{-6lI}{L} & lm \left( \frac{J}{2(1+\nu)} - 4I \right) & \left( \frac{m^2J}{2(1+\nu)} + 4l^2I \right) & \frac{6lI}{L} & -lm \left( \frac{J}{2(1+\nu)} + 2I \right) & \left( \frac{-m^2J}{2(1+\nu)} + 2l^2I \right) \\ \frac{-12I}{L^2} & \frac{-6mI}{L} & \frac{6lI}{L} & \frac{12I}{L^2} & \frac{-6mI}{L} & \frac{6lI}{L} \\ \frac{6mI}{L} & \left( \frac{-l^2J}{2(1+\nu)} + 2m^2I \right) & -lm \left( \frac{J}{2(1+\nu)} + 2I \right) & \frac{-6mI}{L} & \left( \frac{l^2J}{2(1+\nu)} + 4m^2I \right) & lm \left( \frac{J}{2(1+\nu)} - 4I \right) \\ \frac{-6lI}{L} & -lm \left( \frac{J}{2(1+\nu)} + 2I \right) & \left( \frac{-m^2J}{2(1+\nu)} + 2l^2I \right) & \frac{6lI}{L} & lm \left( \frac{J}{2(1+\nu)} - 4I \right) & \left( \frac{m^2J}{2(1+\nu)} + 4l^2I \right) \end{bmatrix}$$

**1.6. Three nodes Plane Elasticity Triangular element** (Élément triangulaire d'élasticité plane à trois nœuds)

$$[k] = [B]^T [C] [B] \Delta t$$

Where:  $\Delta$  is the area of the triangular element and  $t$  is its thickness

$$[B] = \frac{1}{2\Delta} \begin{bmatrix} y_2 - y_3 & 0 & y_3 - y_1 & 0 & y_1 - y_2 & 0 \\ 0 & x_3 - x_2 & 0 & x_1 - x_3 & 0 & x_2 - x_1 \\ x_3 - x_2 & y_2 - y_3 & x_1 - x_3 & y_3 - y_1 & x_2 - x_1 & y_1 - y_2 \end{bmatrix}$$

$$[C] = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \text{ in the plane stresses case}$$

$$[C] = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix} \text{ in the plane deformations case}$$

### 1.7 Tetrahedral element (Volumic four nodes element): (Elément tétraédrique à 4 nœuds)

$$[k] = V[B]^T[C][B]$$

$$[B] = \begin{bmatrix} b_1 & 0 & 0 & b_2 & 0 & 0 & b_3 & 0 & 0 & b_4 & 0 & 0 \\ 0 & c_1 & 0 & 0 & c_2 & 0 & 0 & c_3 & 0 & 0 & c_4 & 0 \\ 0 & 0 & d_1 & 0 & 0 & d_2 & 0 & 0 & d_3 & 0 & 0 & d_4 \\ 0 & d_1 & c_1 & 0 & d_2 & c_2 & 0 & d_3 & c_3 & 0 & d_4 & c_4 \\ d_1 & 0 & b_1 & d_2 & 0 & b_2 & d_3 & 0 & b_3 & d_4 & 0 & b_4 \\ c_1 & b_1 & 0 & c_2 & b_2 & 0 & c_3 & b_3 & 0 & c_4 & b_4 & 0 \end{bmatrix}$$

$$[C] = \frac{E}{(1-2\nu)(1+\nu)} \begin{bmatrix} 1-\nu & \nu & \nu & 0 & 0 & 0 \\ \nu & 1-\nu & \nu & 0 & 0 & 0 \\ \nu & \nu & 1-\nu & 0 & 0 & 0 \\ 0 & 0 & 0 & (1-2\nu)/2 & 0 & 0 \\ 0 & 0 & 0 & 0 & (1-2\nu)/2 & 0 \\ 0 & 0 & 0 & 0 & 0 & (1-2\nu)/2 \end{bmatrix}$$